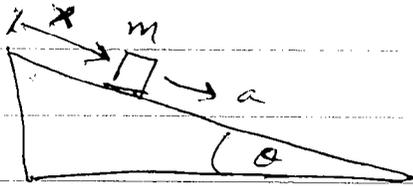


⑤ block on inclined plane



coordinate  $x =$  dist from top of incline

$$T = \frac{1}{2} m \dot{x}^2$$

$$U = -mgx \sin \theta$$

(if take origin at  $x=0$ , then height  $h = -x \sin \theta$ )

$$L = \frac{1}{2} m \dot{x}^2 + mgx \sin \theta$$

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{x}} \right) - \frac{\partial L}{\partial x} = \frac{d}{dt} (m \dot{x}) - mg \sin \theta = 0$$

$$\Rightarrow \ddot{x} = g \sin \theta \quad \text{acceleration down the plane}$$

same result as Newton, but did not need to worry about normal force! Did not need to project  $\vec{F}_g$  into parallel or perpendicular components!

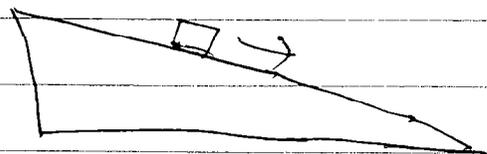
Note: In the last two examples we saw that the Lagrangian approach can be very helpful when there is an unyielding constraint.

④ pendulum



constraint is radial dist = const

⑤ block on plane

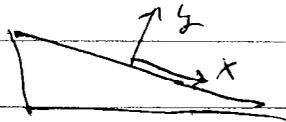


constraint is - block stays on surface of plane

In both cases we can think that there are really two degrees of freedom, and one constraint

pendulum: degrees of freedom =  $\theta, r$   
constraint  $r = l$  constant

Inclined plane: degrees of freedom =  $x, y$   
constraint  $y = 0$



We chose our generalized coordinates so that the constraint eliminated one coord, and the Lagrangian became a function of just the one remaining independent coordinate.

~~More generally, if there are  $N$  degrees of freedom and  $m$  <sup>holonomic</sup> constraints between them, then there are  $S = N - m$  independent degrees of freedom. One can always in principle use the constraints to ~~eliminate~~ <sup>rewrite</sup>  $m$  degrees of freedom, in terms of the others.~~

More generally, if there are  $N$  degrees of freedom, and  $m$  holonomic constraints, then there are  $S = N - m$  independent degrees of freedom. We can then choose  $S$  independent generalized coordinates  $q_i, i=1, \dots, S$ , such that we can write all the  $N$  degrees of freedom in terms of these

ie  $X_k = X_k(q_1, \dots, q_s) \quad k=1, \dots, N$

and  $\dot{X}_k$  in terms of the  $\dot{q}_i$

and then express the Lagrangian in terms of  $\mathcal{L}[q_i, \dot{q}_i; t]$

The Lagrange equations

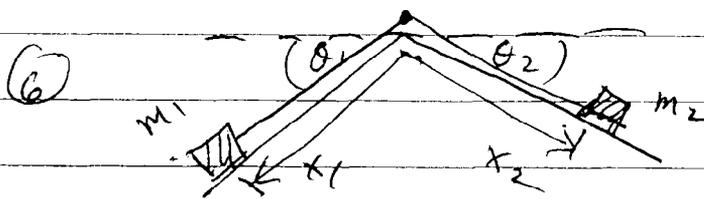
$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{q}_i} \right) - \frac{\partial \mathcal{L}}{\partial q_i} = 0$$

then give  $s$  equations for the  $s$  independent degrees of freedom.

The trick is to choose the  $q_i$  to make the resulting Lagrange eqns as simple as possible.

more examples

two masses tied together on a frictionless inclined plane



constraint:  $x_1 + x_2 = l$  length of rope

$$T = \frac{1}{2} m_1 \dot{x}_1^2 + \frac{1}{2} m_2 \dot{x}_2^2$$

$$U = -m_1 g x_1 \sin \theta_1 - m_2 g x_2 \sin \theta_2$$

since height  $h_1 = -x_1 \sin \theta_1$ ,  $h_2 = -x_2 \sin \theta_2$

use the constraint to eliminate  $x_2$  in terms of  $x_1$ ,  
then write  $\mathcal{L}$  in terms of  $x_1$

$$x_2 = l - x_1 \Rightarrow \dot{x}_2 = -\dot{x}_1$$

$$T = \frac{1}{2}(m_1 + m_2)\dot{x}_1^2$$

$$\begin{aligned} U &= -m_1 g x_1 \sin \theta_1 - m_2 g (l - x_1) \sin \theta_2 \\ &= -g(m_1 \sin \theta_1 - m_2 \sin \theta_2)x_1 - m_2 g l \sin \theta_2 \end{aligned}$$

↑ a constant

(Note: adding a constant to  $\mathcal{L}$  does not change the  
Lagrange equations of motion)

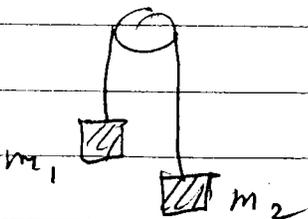
$$\mathcal{L} = \frac{1}{2}(m_1 + m_2)\dot{x}_1^2 + g(m_1 \sin \theta_1 - m_2 \sin \theta_2)x_1 - m_2 g l \sin \theta_2$$

$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{x}_1} \right) - \frac{\partial \mathcal{L}}{\partial x_1} = \frac{d}{dt} \left( (m_1 + m_2)\dot{x}_1 \right) - g(m_1 \sin \theta_1 - m_2 \sin \theta_2) = 0$$

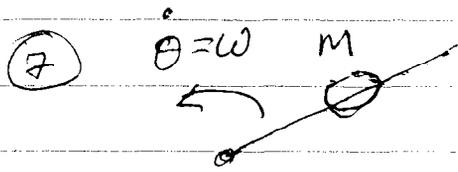
$$\Rightarrow \ddot{x}_1 = \frac{g(m_1 \sin \theta_1 - m_2 \sin \theta_2)}{m_1 + m_2}$$

acceleration is zero if  $m_1 \sin \theta_1 = m_2 \sin \theta_2$

Atwoods machine -  $\theta_1 = \theta_2 = \frac{\pi}{2}$



$$\ddot{x}_1 = \frac{g(m_1 - m_2)}{m_1 + m_2}$$



sliding bead of mass  $m$   
on a rod that rotates  
with angular speed  $\omega$ .

We did this earlier when we discussed velocity  
& acceleration in polar coordinates.

coordinates:  $r, \theta$       constraint  $\dot{\theta} = \omega$  fixed

$$T = \frac{1}{2} m \dot{r}^2 + \frac{1}{2} m (r \dot{\theta})^2 \quad U = 0$$

$$\mathcal{L} = \frac{1}{2} m (\dot{r}^2 + r^2 \dot{\theta}^2) = \frac{1}{2} m (\dot{r}^2 + r^2 \omega^2)$$

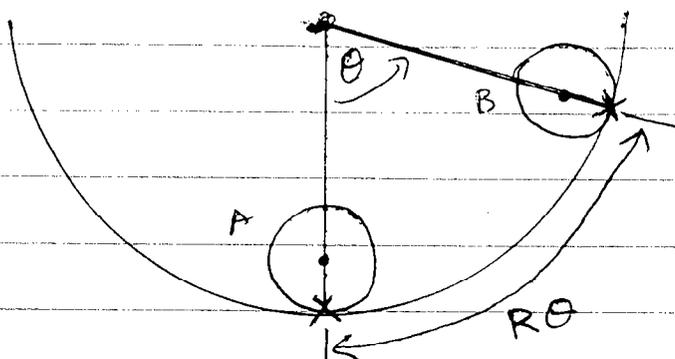
$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{r}} \right) - \frac{\partial \mathcal{L}}{\partial r} = \frac{d}{dt} (m \dot{r}) - m r \omega^2 = 0$$

$$\ddot{r} = \omega^2 r$$

$$r(t) = r(0) e^{\omega t}$$

Same solution as found earlier

⑧ prob 7-3



sphere of radius  $r$   
rolling on cylinder of  
Radius  $R$

Suppose that in rolling from position A to B, the point "x" on the sphere that is in contact with the cylinder at A, is once again in contact with the cylinder at B.

$\Rightarrow$  distance rolled  $R\theta = 2\pi r$   
in this process, the sphere has rotated through  
an angle  $(2\pi - \theta) = \phi$

In general, if distance rolled is some fraction  $s$  of  $2\pi r$ ,  
i.e.  $R\theta = 2\pi s r$ , then the sphere has rotated  
through  $(2\pi s - \theta) = \phi \Rightarrow 2\pi s = \phi + \theta$

$$\Rightarrow \boxed{R\theta = (\theta + \phi)r}$$

constraint relating angle  $\theta$   
to angle  $\phi$  of rotation of  
sphere about its center

We will choose  $\theta$  as the generalized coordinate in  
which to do the problem.

$$\frac{(R-r)\theta}{r} = \phi \leftarrow \text{constraint}$$

$$T = \frac{1}{2} m (R-p)^2 \dot{\theta}^2 + \frac{1}{2} I \dot{\phi}^2$$

$\uparrow$  center of mass KE  
 substitute for  $\dot{\phi}$

$\uparrow$  rotational KE

$$T = \frac{1}{2} m (R-p)^2 \dot{\theta}^2 + \frac{1}{2} \frac{2}{5} m p^2 \frac{(R-p)^2}{p^2} \dot{\theta}^2$$

$$T = \frac{1}{2} \frac{7}{5} m (R-p)^2 \dot{\theta}^2$$

If place origin at the center of the cylinder, the potential energy of gravity is

$$U = -mg(R-p) \cos \theta$$

$$\Rightarrow \mathcal{L} = \frac{1}{2} \frac{7}{5} m (R-p)^2 \dot{\theta}^2 + mg(R-p) \cos \theta$$

$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{\theta}} \right) - \frac{\partial \mathcal{L}}{\partial \theta} = \frac{7}{5} m (R-p)^2 \ddot{\theta} + mg(R-p) \sin \theta = 0$$

$$\Rightarrow \boxed{\ddot{\theta} + \frac{5g \sin \theta}{7(R-p)} = 0} \approx \ddot{\theta} + \frac{5g}{7(R-p)} \theta = 0$$

for small oscillations

same as a pendulum with angular freq

$$\omega = \sqrt{\frac{5g}{7(R-p)}}$$

oscillates at. freq  $\omega = \sqrt{\frac{g}{R-p + \frac{I}{m(R-p)} \left(\frac{R}{p}\right)^2}}$

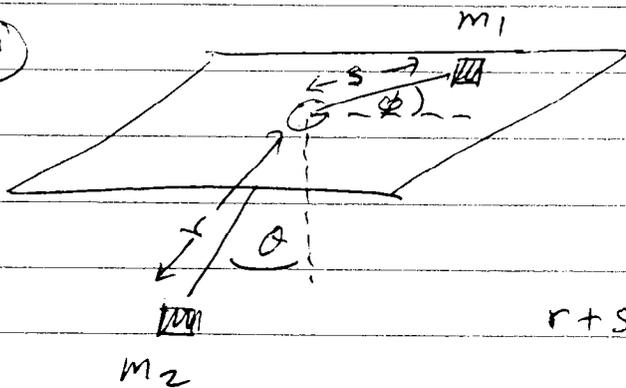
Uniform  
For a sphere,  $I = \frac{2}{5} m R^2$

$$\omega = \sqrt{\frac{g}{R-p + \frac{2}{5} \frac{R^2}{(R-p)}}$$

if  $g \ll R$  then

$$\omega \approx \sqrt{\frac{g}{R + \frac{2}{5} R}} = \sqrt{\frac{5g}{7R}}$$

9



connected mass, one on plane, other hanging

$r + s = l$  constraint

$$T = \frac{1}{2} m_1 (\dot{s}^2 + (s\dot{\phi})^2) + \frac{1}{2} m_2 (\dot{r}^2 + (r\dot{\theta})^2)$$

$$U = -m_2 g r \cos \theta$$

use  $s = l - r$

$$\begin{aligned} \mathcal{L} = T - U = & \frac{1}{2} m_1 (\dot{r}^2 + (l-r)^2 \dot{\phi}^2) + \frac{1}{2} m_2 (\dot{r}^2 + r^2 \dot{\theta}^2) \\ & + m_2 g r \cos \theta \end{aligned}$$

$$\frac{\partial \mathcal{L}}{\partial \dot{r}} = m_1 \dot{r} + m_2 \dot{r} = (m_1 + m_2) \dot{r}$$

$$\frac{\partial \mathcal{L}}{\partial r} = -m_1 (l-r) \dot{\phi}^2 + m_2 r \dot{\theta}^2 + m_2 g \cos \theta$$

$$\frac{\partial \mathcal{L}}{\partial \dot{\phi}} = m_1 (l-r)^2 \dot{\phi} \quad \frac{\partial \mathcal{L}}{\partial \phi} = 0$$

$$\frac{\partial \mathcal{L}}{\partial \dot{\theta}} = m_2 r^2 \dot{\theta} \quad \frac{\partial \mathcal{L}}{\partial \theta} = -m_2 g r \sin \theta$$

Lagrange equs:  $\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{\phi}} \right) - \frac{\partial \mathcal{L}}{\partial \phi} = 0$

$$\Rightarrow m_1 \frac{d}{dt} [(l-r)^2 \dot{\phi}] = 0 \quad \Rightarrow \text{Conservation of } \hat{z} \text{ component of angular momentum.}$$

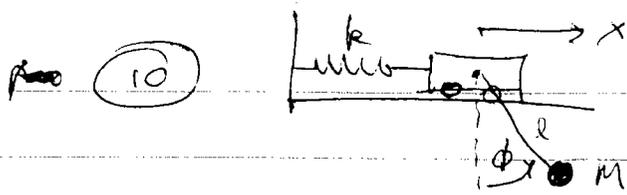
$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{\theta}} \right) - \frac{\partial \mathcal{L}}{\partial \theta} = 0 \Rightarrow \frac{d}{dt} (m_2 r^2 \dot{\theta}) = -m_2 g r \sin \theta$$

change of angular momentum in vertical plane = Torque from gravity

$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{r}} \right) - \frac{\partial \mathcal{L}}{\partial r} = 0 \Rightarrow$$

$$\frac{d}{dt} (m_1 \dot{r} + m_2 \dot{r}) = m_1 (l-r) \dot{\phi}^2 + m_2 r \dot{\theta}^2 - m_2 g \cos \theta$$

$$(m_1 + m_2) \ddot{r} + m_1 (l-r) \dot{\phi}^2 - m_2 r \dot{\theta}^2 - m_2 g \cos \theta = 0$$



pendulum of mass  $m$   
attached to massless cart  
on spring.

coordinates  $x, \phi$

$$T = \frac{1}{2} m \vec{v}^2 \quad \text{where } \vec{v} \text{ is velocity of mass } m.$$

Choose coordinates  $x', y'$  centered on cart. Then mass is at

$$\left. \begin{aligned} x' &= l \sin \phi \\ y' &= -l \cos \phi \end{aligned} \right\} \Rightarrow \left. \begin{aligned} \dot{x}' &= l \cos \phi \dot{\phi} \\ \dot{y}' &= l \sin \phi \dot{\phi} \end{aligned} \right\} \begin{array}{l} \text{velocity of mass} \\ \text{relative to} \\ \text{center of cart} \end{array}$$

to get velocity of mass in lab frame, we need  
addition of velocities

$$v_x = \dot{x} + \dot{x}' = \dot{x} + l \dot{\phi} \cos \phi$$

$$v_y = \dot{y}' = l \dot{\phi} \sin \phi$$

$$T = \frac{1}{2} m \left( (\dot{x} + l \dot{\phi} \cos \phi)^2 + (l \dot{\phi} \sin \phi)^2 \right)$$

$$= \frac{1}{2} m \left( \dot{x}^2 + l^2 \dot{\phi}^2 + 2 \dot{x} l \dot{\phi} \cos \phi \right)$$

Potential energy  $U = \frac{1}{2} k x^2 + m g y$

$$= \frac{1}{2} k x^2 - m g l \cos \phi$$

$$\mathcal{L} = T - U = \frac{1}{2} m \dot{x}^2 + \frac{1}{2} m l^2 \dot{\phi}^2 + m \dot{x} l \dot{\phi} \cos \phi - \frac{1}{2} k x^2 + m g l \cos \phi$$

Lagrange equ for  $x$ :

$$\frac{\partial \mathcal{L}}{\partial x} = -kx, \quad \frac{\partial \mathcal{L}}{\partial \dot{x}} = m\dot{x} + ml\dot{\phi} \cos \phi$$

$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{x}} \right) = m\ddot{x} + ml\ddot{\phi} \cos \phi - ml\dot{\phi}^2 \sin \phi$$

$$\Rightarrow \boxed{m\ddot{x} + ml\ddot{\phi} \cos \phi - ml\dot{\phi}^2 \sin \phi + kx = 0} \quad (1)$$

Lagrange equ for  $\phi$ :

$$\frac{\partial \mathcal{L}}{\partial \phi} = -m\dot{x}l\dot{\phi} \sin \phi - mgl \sin \phi$$

$$\frac{\partial \mathcal{L}}{\partial \dot{\phi}} = ml^2\ddot{\phi} + m\dot{x}l \cos \phi$$

$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{\phi}} \right) = ml^2\ddot{\phi} + m\ddot{x}l \cos \phi - m\dot{x}l\dot{\phi} \sin \phi$$

$$\Rightarrow ml^2\ddot{\phi} + m\ddot{x}l \cos \phi - m\dot{x}l\dot{\phi} \sin \phi + m\dot{x}l\dot{\phi} \sin \phi + mgl \sin \phi = 0$$

$$\Rightarrow \boxed{ml^2\ddot{\phi} + m\ddot{x}l \cos \phi + mgl \sin \phi = 0} \quad (2)$$

need to solve above two differential equation for  $x(t)$  and  $\phi(t)$ .

If small oscillations, then approx  $\sin \phi \approx \phi$ ,  $\cos \phi \approx 1$

$$\Rightarrow \begin{cases} m\ddot{x} + ml\ddot{\phi} - ml\dot{\phi}^2 \phi + kx = 0 \\ m\ddot{x} + ml\ddot{\phi} + mgl\phi = 0 \end{cases} \quad \leftarrow \text{divided (2) by } l$$

subtract

$$mg\phi + ml\dot{\phi}^2 \phi + kx = 0$$

high order in small quantities, so ignore

$$\Rightarrow mg\phi + kx = 0$$

$$\Rightarrow x = \frac{-mg\phi}{k}$$

use substitute into  $m\ddot{x} + m\ell\ddot{\phi} + mg\phi = 0$  to get

$$-\frac{m^2g}{k}\ddot{\phi} + m\ell\ddot{\phi} + mg\phi = 0$$

$$\Rightarrow \ddot{\phi} \left( 1 + \frac{mg}{k\ell} \right) + \frac{g}{\ell}\phi = 0$$

$$\ddot{\phi} + \frac{\left( \frac{g}{\ell} \right)}{1 + \left( \frac{g}{\ell} \right) \left( \frac{m}{k} \right)} \phi = 0$$

pendulum oscillates with freq

$$\omega = \sqrt{\frac{g/\ell}{1 + (g/\ell)(m/k)}} = \sqrt{\frac{g}{\ell + gm/k}}$$

If the cart is fixed, this is equivalent to taking limit of infinitely stiff spring  $k \rightarrow \infty$ . In this case we recover ordinary pendulum result  $\omega = \sqrt{g/\ell}$